- Kinematic Viscosity :
- Kinematic viscosity is defined as the ratio between the dynamic viscosity and density of fluid. t is denoted by v (called nu).

$$v = \frac{\text{Viscosity}}{\text{Density}} = \frac{\mu}{\rho}$$

- Units of kinematic viscosity:
- In SI units: m2/s
- In M.K.S. units: m2/sec.
- In C.G.S. units the kinematic viscosity is also known as stoke (= cm2/sec.)
- One stoke = 10–4 m2/s

Newton's Law of Viscosity

This law states that the shear stress (T) on a fluid element layer is directly proportional to the rate of shear strain. The constant of proportionality is called the co-efficient of viscosity.

$$\tau = \mu \frac{du}{dy}$$

The fluids which follow this law are known as Newtonian fluids.

- Effect of Temperature on Viscosity
- Viscosity is effected by temperature. The viscosity of *liquids decreases* but that of gases increases with *increase in temperature*.
- This is due to the reason that in *liquids* the shear stress is due to the intermolecular cohesion which *decreases* with increase of temperature
- In gases theinter-molecular cohesion is negligible and the shear stress is due to exchange of momentum of the molecules, normal to the direction of motion.
- The molecular activity increases with rise in temperature and so does the viscosity of gas.

For liquids:

For gases:

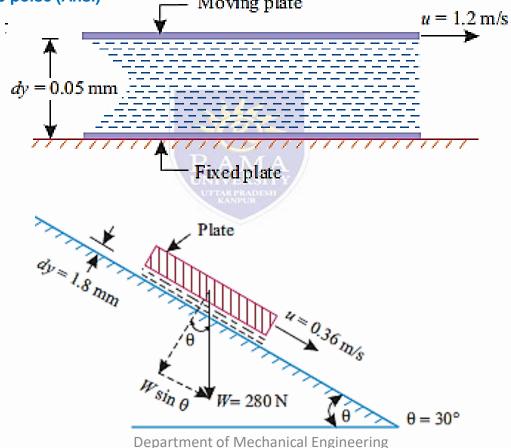
$$\mu_T = Ae^{\beta/T}$$
$$\mu T = \frac{bT^{1/2}}{1 + a/T}$$

where,

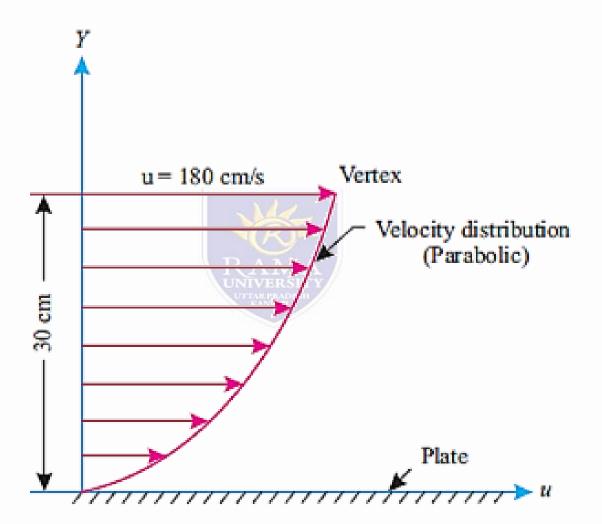
 $\mu_T = \text{Dynamic viscosity at absolute temperature } T,$ $A, \beta = \text{Constants (for a given liquid), and}$ a, b = Constants (for a given gas). Department of Mechanical Engineering 20

Lecture -03 - Fluid Properties

- Effect of Pressure on Viscosity
- The viscosity under ordinary conditions is not appreciably affected by the changes in pressure. However, the viscosity of some oils has been found to increase with increase in pressure.
- A plate 0.05 mm distant from a fixed plate moves at 1.2 m/s and requires a force of 2.2 N/m2 to maintain this speed. Find the viscosity of the fluid between the plates. 9.16 × 10–4 poise (Ans.)
- A plate having an area of 0.6 m2 is sliding down the inclined plane at 30° to the horizontal with a velocity of 0.36 m/s. There is a cushion of fluid 1.8 mm thick between the plane and the plate. Find the viscosity of the fluid if the weight of the plate is 280 N. 11.66 poise (Ans.)



• The velocity distribution of flow over a plate is parabolic with vertex 30 cm from the plate, where the velocity is 180 cm/s. If the viscosity of the fluid is 0.9 N.s/m2 find the velocity gradients and shear stresses at distances of 0, 15 cm and 30 cm from the plate.



• A 400 mm diameter shaft is rotating at 200 r.p.m. in a bearing of length 120 mm. If the thickness of oil film is 1.5 mm and the dynamic viscosity of the oil is 0.7 N.s/m2, determine:

Shaft

- (i) Torque required to overcome friction in bearing;
- (ii) Power utilised in overcoming viscous resistance.
- Assume a linear velocity profile.

Solution. Diameter of the shaft, d = 400 mm = 0.4 m

Speed of the shaft, N = 200 r.p.m. Thickness of the oil film, $t = 1.5 \text{ mm} = 1.5 \times 10^{-3} \text{ m}$ Length of the bearing, l = 120 mm = 0.12 mViscosity, $\mu = 0.7 \text{ N.s/m}^2$

Tangential velocity of the shaft,
$$u = \frac{\pi dN}{60} = \frac{\pi \times 0.4 \times 200}{60} = 4.19 \text{ m/s}$$

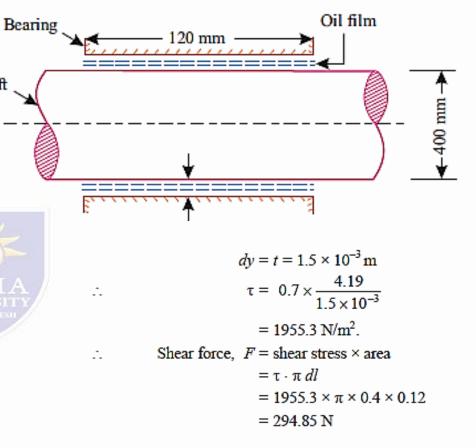
(i) Torque required to overcome friction, T:

We know, $\tau = \mu . \frac{du}{dy}$

where du = change of velocity = u - 0 = 4.19 m/s

(ii) Power utilised, P:

$$P = T \times \frac{2\pi N}{60}$$
 watts, where *T* is in Nm
$$P = 58.97 \times \frac{2\pi \times 200}{60} = 1235$$
 W or **1.235** kW (Ans.



Hence, viscous torque =
$$F \times d/2 = 294.85 \times \frac{0.4}{2}$$

= 58.97 Nm (Ans.)

or

or

A circular disc of diameter D is slowly rotated in a liquid of large viscosity (μ) at a small distance (h) from a fixed surface. Derive an expression of torque (T) necessary to maintain an angular velocity (ω).

Consider an elementary ring of disc at radius r and having a width dr. Linear velocity at this radius is or.

Shear stress,
$$\tau = \mu \frac{du}{dy}$$

Torque = shear stress × area × r
= $\tau \times 2\pi r \, dr \times r$
= $\mu \frac{du}{dy} \times 2\pi r^2 \times dr$

Assuming the gap h to be small so that the velocity distribution may be assumed linear.

