

# FACULTY OF ENGINEERING AND TECHNOLOGY

**Department of Mechanical Engineering** 

# **MEPS102:Strength of Material** Lecture 35 Topic:Castigliano's Theorem Instructor:

### **Castigliano's Theorem**

It provides a means for finding the deflections of a structure from the strain energy of the structure

The partial derivative of the strain energy of a structure with respect to any load is equal to the displacement corresponding to that load

$$\delta_i = \frac{\partial U}{\partial P_i}$$

The theorem can be applied to any kind of loading. The important requirements are that the structure be linearly elastic and that the principle of superposition be applicable. Also, note that the strain energy must be expressed as a function of the loads (and not as a function of the displacements), a condition which is implied in the theorem itself, since the partial derivative is taken with respect to a load.

# Application of Castigliano's Theorem

$$M = -Px - M_{0}$$

$$U = \int_{0}^{L} \frac{M^{2} dx}{2EI} = \frac{1}{2EI} \int_{0}^{L} (-Px - M_{0})^{2} dx$$

$$= \frac{P^{2}L^{3}}{6EI} + \frac{PM_{0}L^{2}}{2EI} + \frac{M_{0}^{2}L}{2EI}$$

$$\delta_{A} = \frac{\partial U}{\partial P} = \frac{PL^{3}}{3EI} + \frac{M_{0}L^{2}}{2EI}$$

$$A = \frac{M_{0}L^{2}}{\delta_{C}}$$

$$A = \frac{\partial U}{\delta_{C}} = \frac{PL^{3}}{2EI} + \frac{M_{0}L^{2}}{2EI}$$

$$A = \frac{\partial U}{\delta_{C}} = \frac{PL^{3}}{2EI} + \frac{M_{0}L^{2}}{2EI}$$

$$\theta_A = \frac{\partial U}{\partial M_0} = \frac{PL^2}{2EI} + \frac{M_0 L}{EI}$$

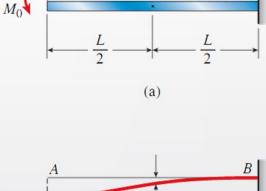
# Use of a Fictitious Load

- The only displacements that can be found from Castigliano's theorem are those that correspond to loads acting on the structure.
- ✓ If we wish to calculate a displacement at a point on a structure where there is no load, then a **fictitious load** corresponding to the desired displacement must be applied to the structure.
- We can then determine the displacement by evaluating the strain energy and taking the partial derivative with respect to the fictitious load.
- The result is the displacement produced by the actual loads and the fictitious load acting simultaneously.
- ✓ By setting the fictitious load equal to zero, we obtain the displacement produced only by the actual loads.

### Use of a Fictitious Load

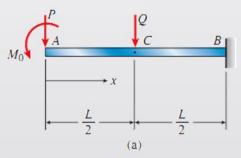
✓ We wish to find out deflection at C

$$\begin{split} M &= -Px - M_0 \qquad \left( 0 \le x \le \frac{L}{2} \right) \\ M &= -Px - M_0 - Q\left(x - \frac{L}{2}\right) \qquad \left( \frac{L}{2} \le x \le L \right) \\ U_{AC} &= \int_0^{L/2} \frac{M^2 dx}{2EI} = \frac{1}{2EI} \int_0^{L/2} (-Px - M_0)^2 dx \\ &= \frac{P^2 L^3}{48EI} + \frac{P M_0 L^2}{8EI} + \frac{M_0^2 L}{4EI} \end{split}$$



 $\delta_C$ 

Beam with a fictitious load Q





#### Use of a Fictitious Load

$$U_{CB} = \int_{L2}^{L} \frac{M^{2} dx}{2EI} = \frac{1}{2EI} \int_{L/2}^{L} \left[ -Px - M_{0} - Q\left(x - \frac{L}{2}\right) \right]^{2} dx$$

$$= \frac{7P^{2}L^{3}}{48EI} + \frac{3PM_{0}L^{2}}{8EI} + \frac{5PQL^{3}}{48EI} + \frac{M_{0}^{2}L}{4EI} + \frac{M_{0}Q^{2}}{8EI} + \frac{Q^{2}L^{3}}{48EI}$$

$$U = U_{AC} + U_{CB}$$

$$= \frac{P^{2}L^{3}}{6EI} + \frac{PM_{0}L^{2}}{2EI} + \frac{5PQL^{3}}{48EI} + \frac{M_{0}^{2}L}{2EI} + \frac{M_{0}QL^{2}}{8EI} + \frac{Q^{2}L^{3}}{48EI}$$

$$\left(\delta_{C}\right)_{0} = \frac{\partial U}{\partial Q} = \frac{5PL^{3}}{48EI} + \frac{M_{0}L^{2}}{8EI} + \frac{QL^{3}}{24EI}$$

$$\delta_{C} = \frac{5PL^{3}}{48EI} + \frac{M_{0}L^{2}}{8EI}$$

This method is sometimes called the dummy-load method, because of the introduction of a fictitious, or dummy, load.

# Modified Castigliano's theorem (Unit Load Method)

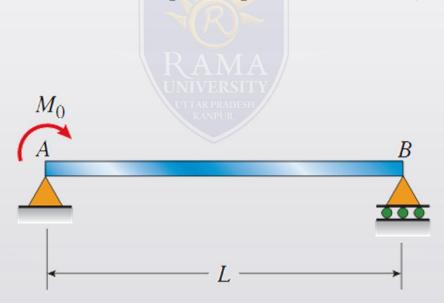
The use of Castigliano's theorem for determining beam deflections may lead to lengthy integrations, if the bending moment expression has three terms, its square may have as many as six terms, each of which must be integrated.

 we can bypass the step of finding the strain energy by differentiating before integrating.

$$\delta_{i} = \frac{\partial U}{\partial P_{i}} = \frac{\partial}{\partial P_{i}} \int \frac{M^{2}}{2EI} dx = \int \frac{M}{EI} \left(\frac{\partial M}{\partial P_{i}}\right) dx$$

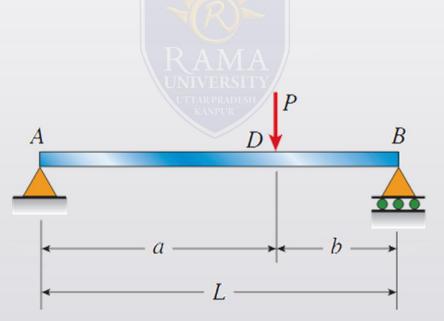
**9.9-1** A simple beam *AB* of length *L* is loaded at the lefthand end by a couple of moment  $M_0$  (see figure).

Determine the angle of rotation  $\theta_A$  at support A. (Obtain the solution by determining the strain energy of the beam and then using Castigliano's theorem.)



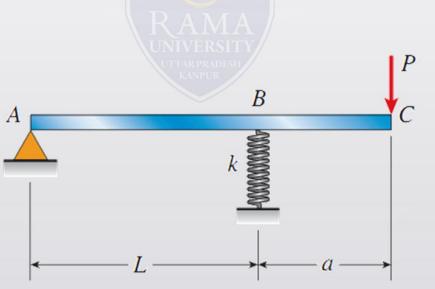
**9.9-2** The simple beam shown in the figure supports a concentrated load P acting at distance a from the left-hand support and distance b from the right-hand support.

Determine the deflection  $\delta_D$  at point *D* where the load is applied. (Obtain the solution by determining the strain energy of the beam and then using Castigliano's theorem.)



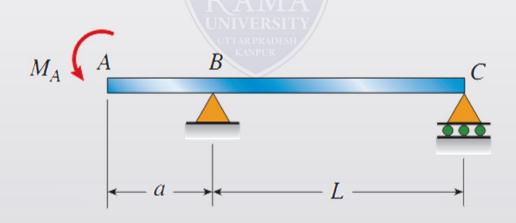
**9.9-11** An overhanging beam *ABC* rests on a simple support at *A* and a spring support at *B* (see figure). A concentrated load *P* acts at the end of the overhang. Span *AB* has length *L*, the overhang has length *a*, and the spring has stiffness k.

Determine the downward displacement  $\delta_C$  of the end of the overhang. (Obtain the solution by using the modified form of Castigliano's theorem.)



**9.9-10** An overhanging beam *ABC* is subjected to a couple  $M_A$  at the free end (see figure). The lengths of the overhang and the main span are *a* and *L*, respectively.

Determine the angle of rotation  $\theta_A$  and deflection  $\delta_A$  at end A. (Obtain the solution by using the modified form of Castigliano's theorem.)



**9.9-7** The cantilever beam ACB shown in the figure is subjected to a uniform load of intensity q acting between points A and C.

Determine the angle of rotation  $\theta_A$  at the free end A. (Obtain the solution by using the modified form of Castigliano's theorem.)

