

FACULTY OF ENGINEERING AND TECHNOLOGY

Department of Mechanical Engineering

MEPS102:Strength of Material

Lecture 8

Topic: The Torsion Formula, Non-uniform torsion, Stresses and strains in pure shear

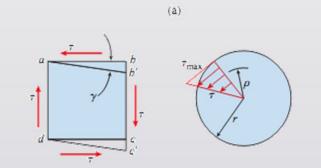
Instructor:

Aditya Veer Gautam

The Torsion Formula

✓ In this lecture we will determine the relationship between the shear stresses and the torque T.

The distribution of the shear stresses acting on a cross section is pictured in below Because these stresses act continuously around the cross section, they have a resultant in the form of a moment—a moment equal to the torque T acting on the bar



The Torsion Formula

- ✓ To determine this resultant, we con-sider an element of area dA located at radial distance r from the axis of the bar. The shear force acting on this element is equal to τdA , where τ is the shear stress at radius r.
- The moment of this force about the axis of the bar is equal to the force times its distance from the center, or $\tau \rho dA$.
- ✓ There elemental moment is

$$dM = \tau dA = \frac{\tau_{max}}{r} \rho^2 dA$$

The resultant moment (torque T) is

$$T = \int_{A} dM = \frac{\tau_{max}}{r} \int_{A} \rho^{2} dA = \frac{\tau_{max}}{r} I_{P}$$

Where $I_P = \int_A \rho^2 dA$, is polar moment of inertia

Polar moments of inertia have units of length to the fourth power

Max Shear Stress

 \checkmark An expression for the maximum shear stress is given by

$$\tau_{max} = \frac{Tr}{I_p}$$

✓ For a prismatic bars of solid circular cross section

$$r = \frac{d}{2} \text{ and } I_P = \frac{\pi d^4}{32}$$
$$\tau_{max} = \frac{16T}{\pi d^3}$$

 \checkmark Shear Stress at a distance of r from the center of bar is given by

$$\tau = \frac{\rho}{r} \tau_{max} = \frac{T\rho}{I_p}$$

Angle of Twist

 The angle of twist of a bar of linearly elastic material can now be related to the applied torque T

$$\theta = \frac{T}{GI_p}$$

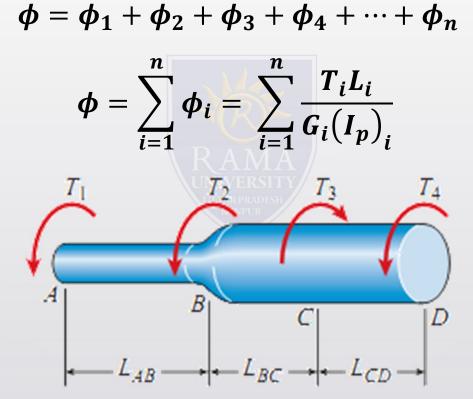
GI_p is known as Torsional Rigidity

$$\phi = \frac{TL}{GI_p}$$

 $\frac{GI_p}{L} = k_T \text{ is called torsional stiffness}$

 θ has units of radians per unit of length

Equations for the torsion of circular bars and tubes cannot be used for bars of other shapes. Noncircular bars, such as rectangular bars and bars having lshaped cross sections, their cross sections do not remain plane and their maximum stresses are not located at the farthest distances from the midpoints of the cross sections Case 1: Bar consisting of prismatic segments with constant torque throughout each segment



Non-uniform torsion

Case 2: Bar with continuously varying cross sections and constant torque

Non-uniform torsion

Case 3: Bar with continuously varying cross sections and continuously varying torque

$$\phi = \int_{0}^{L} d\phi = \int_{0}^{L} \frac{T(x)dx}{GI_{p}(x)}$$

$$T_{A}$$

$$T_{A}$$

$$T_{A}$$

$$T_{A}$$

$$T_{A}$$

$$T_{A}$$

$$T_{A}$$

$$T_{B}$$

Stresses and Strain in Pure Shear

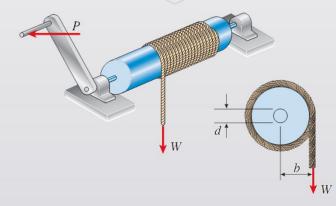
- ✓ by considering a stress element abcd cut between two cross sections of a bar in torsion . This element is in a state of pure shear, because the only stresses acting on it are the shear stresses τ on the four side face
- The directions of these shear stresses depend upon the directions of the applied torques T
- ✓ This same state of stress exists for a similar element cut from the interior of the bar, except that the magnitudes of the shear stresses are smaller because the radial distance to the element is smaller

$$T$$

3.3-1 A prospector uses a hand-powered winch (see figure) to raise a bucket of ore in his mine shaft. The axle of the winch is a steel rod of diameter d = 15 mm. Also, the distance from the center of the axle to the center of the lifting rope is b = 100 mm.

(a) If the weight of the loaded bucket is W = 400 N, what is the maximum shear stress in the axle due to torsion?

(b) If the maximum bucket load is 510 N and the allowable shear stress in the axle is 65 MPa, what is the minimum permissible axle diameter?



225 mm

P = 100

d = 12 mm

3.3-3 While removing a wheel to change a tire, a driver applies forces P = 100 N at the ends of two of the arms of a lug wrench (see figure). The wrench is made of steel with shear modulus of elasticity G = 78 GPa. Each arm of the wrench is 255 mm long and has a solid circular cross section of diameter d = 12 mm.

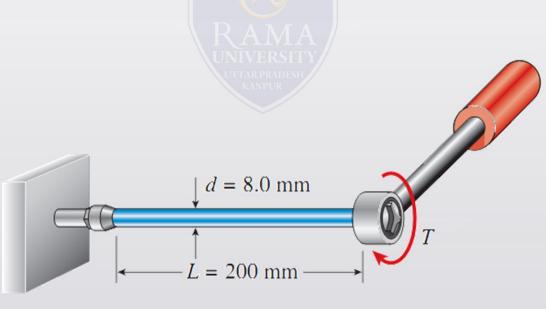
(a) Determine the maximum shear stress in the arm that is turning the lug nut (arm A).

(b) Determine the angle of twist (in degrees) of this same arm.

3.3-6 The steel shaft of a socket wrench has a diameter of 8.0 mm. and a length of 200 mm (see figure).

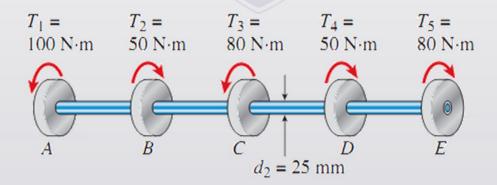
If the allowable stress in shear is 60 MPa, what is the maximum permissible torque T_{max} that may be exerted with the wrench?

Through what angle ϕ (in degrees) will the shaft twist under the action of the maximum torque? (Assume G = 78 GPa and disregard any bending of the shaft.)



3.4-5 A hollow tube *ABCDE* constructed of monel metal is subjected to five torques acting in the directions shown in the figure. The magnitudes of the torques are $T_1 = 100 \text{ N} \cdot \text{m}$, $T_2 = T_4 = 50 \text{ N} \cdot \text{m}$, and $T_3 = T_5 = 80 \text{ N} \cdot \text{m}$. The tube has an outside diameter of $d_2 = 25 \text{ mm}$. The allowable shear stress is 80 MPa and the allowable rate of twist is 6°/m.

Determine the maximum permissible inside diameter d_1 of the tube.



3.4-8 A tapered bar *AB* of solid circular cross section is twisted by torques *T* (see figure). The diameter of the bar varies linearly from d_A at the left-hand end to d_B at the right-hand end.

(a) Confirm that the angle of twist of the tapered bar is

$$\phi = \frac{32TL}{3\pi G(d_B - d_A)} \left(\frac{1}{d_A^3} - \frac{1}{d_B^3}\right)$$

(b) For what ratio d_B/d_A will the angle of twist of the tapered bar be one-half the angle of twist of a prismatic bar of diameter d_A ? (The prismatic bar is made of the same material, has the same length, and is subjected to the same torque as the tapered bar.)

